

# A Time-Stepping Scheme for Quasistatic Multibody Systems \*

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## Abstract

*Two new instantaneous-time models for predicting the motion and contact forces of three-dimensional, quasistatic multi-rigid-body systems are developed; one linear and one nonlinear. The nonlinear characteristic is the result of retaining the usual quadratic friction cone in the model. Discrete-time versions of these models provide the first time-stepping methods for such systems. As a first step to understanding their usefulness in simulation and manipulation planning, a theorem for solution uniqueness is presented along with simulation results for a simple example.*

## 1 Introduction

Robots are primarily passive observers and simple electronic companions in the unstructured environments that exist outside factories. This is true despite the fact that, as a society, enormous productivity gains could be accrued by expanding the skills of robots to include manipulation tasks; tasks that cannot be accomplished without making and breaking contact between the robot and physical objects in a controlled fashion. Nearly one million household robots are in use world wide today, but these robots cannot perform manipulation tasks autonomously. Even the highly capable Sony QRIO robot cannot do such tasks, although it can walk and dance on sloping terrain. Currently, robotic dexterous manipulation can only be performed in unstructured environments by tele-operation, and it is well-known that this approach is exceedingly slow and places great demand on the operator. As a result, autonomous grasping controllers are being developed, but are still of limited capability [10].

Manipulation tasks can be partitioned into two classes: dynamic and quasistatic. The former class is by far the

broadest and includes high-speed assembly, juggling, and running. However, despite being narrower, the latter class includes a large number of important tasks, such as low-speed assembly, static grasping, walking using tripods of support. The ability to perform tasks in this class motivate the study presented below.

### 1.1 Background

The field of multibody dynamics has been of interest since DaVinci's work in the 1490's. His interest stemmed from a desire to build better machines. About 250 years later, some basic "laws" of mechanics had been developed by Newton and Coulomb, which allowed one to formulate an instantaneous-time mathematical model of dynamic multi-rigid-body systems. This model is composed of the Newton-Euler equation, Coulomb's friction law, and non-penetration constraints with unknown contact forces and body accelerations. In 1895, Painleve was the first to discover that this model does not always admit a solution (this is sometimes referred to as Painleve's paradox) [12]. Existence and uniqueness questions were studied for more general systems after the advent of complementarity theory in the 1960's [5]. In particular, Lötstedt found that when friction is absent, the model can be cast as a linear complementarity problem (LCP) that possesses a property known as "w-uniqueness." The physical interpretation of this property is that the body accelerations are unique, but the contact forces are not [8, 9]. Since Lötstedt's work, existence and uniqueness properties have been extended to include limited results for systems with friction [13, 17]. Specifically, solution existence can only be guaranteed if the friction coefficients at the contacts are below some threshold value, which unfortunately, is exceedingly difficult to compute and is sensitive to the contact geometry.

Because of the weakness of the existence and uniqueness results, it is not advisable to apply standard time-stepping methods directly to the instantaneous model [6, 7]. A superior approach is to derive a discrete-time model written in terms of the unknown contact impulses and body

\*This work was supported by the National Science Foundation under grants 0139701 (DMS-FRG), 0413227 (IIS-RCV), and 0420703 (MRI) and by Rensselaer Polytechnic Institute.

velocities [1, 16]. The Stewart-Trinkle formulation results in an LCP that incorporates constraint stabilization and is nearly always solvable [16]. Moreover, when a solution exists, it can be found using Lemke’s algorithm [5]. If a solution to the Stewart-Trinkle LCP does not exist, one simply drops the constraint stabilization term, yielding the Anitescu-Potra LCP for which a solution always exists and can be found by Lemke’s algorithm [1]. One might wonder why a solution always exists for the discrete-time model when the same is not true for the instantaneous-time model. An intuitive explanation is that since the discrete-time model is written in terms of impulses (applied over the current time-step), it implicitly expands the space of contact forces to include infinite impulses. This is consistent with the resolution to Painleve’s paradox offered by Mason and Wang [11].

Since time-stepping methods are now reasonably well developed for dynamic rigid body systems [1, 2, 3, 15, 16], one might wonder why the focus of this paper is on quasistatic models. The reasons spurs from an interest in the development of planning algorithms. Dynamic systems “live” in state space, which has twice the dimension of configuration space, in which quasistatic systems “live.” Secondly, quasistatic systems move slowly, so inertial, Coriolis, and impulsive forces are absent. Finally, in some cases, a quasistatic manipulation plan can serve as a good initial guess for a dynamic plan.

In previous work, Pang *et al.* [14] formulated an instantaneous-time planar quasistatic model as an uncoupled complementarity problem (UCP) and developed a bilinear programming algorithm to solve it. In this paper, the work is extended to three dimensions, a simple time-stepping scheme is derived, and new solution existence and uniqueness results are given.

## 2 Instantaneous-time models

Let  $q \in \mathfrak{R}^{n_q}$  be the configuration of a system of rigid bodies,  $\nu \in \mathfrak{R}^{n_\nu}$  be the generalized velocity, and  $f(q, t) \in \mathfrak{R}^{n_\nu}$  represent the applied external generalized force, with  $t$  being time. Further, let  $\{\lambda_{in} \geq 0\}_{i=1}^{n_c}$  be the nonnegative normal force at the  $i^{\text{th}}$  contact point, and  $\lambda_{it}$  and  $\lambda_{io}$  be the corresponding orthogonal friction force components. Since a quasistatic system must satisfy equilibrium at all times, the equilibrium equation is needed. It can be written as:

$$0 = W_n(q)\lambda_n + W_t(q)\lambda_t + W_o(q)\lambda_o + f(q, t) \quad (1)$$

where  $\lambda_n, \lambda_t, \lambda_o \in \mathfrak{R}^{n_c}$  are the vectors of normal and friction force components of the contacts (also called wrench intensities),  $W_n, W_t, W_o \in \mathfrak{R}^{(n_c \times n_\nu)}$ , are matrices whose

columns are unit wrenches of the contact normals, and orthogonal tangent plane directions.

The system must also obey a nonpenetration constraint at each contact and a complementarity relationship between the normal component of contact force and the distance function  $\psi_{in}(q, t)$  between the contacting bodies. The linear complementarity constraint is:

$$0 \leq \lambda_n \perp \psi_n(q, t) \geq 0 \quad (2)$$

where  $\psi_n(q, t) \in \mathfrak{R}^{n_\nu}$  is the vector of distance functions with  $i^{\text{th}}$  element given by  $\psi_{in}(q, t)$ , the symbol  $\perp$  implies perpendicularity (*i.e.*,  $\lambda_n \cdot \psi_n = 0$ ). The physical interpretation of equation (2) is that a force may act at contact  $i$  only if the distance between the bodies is zero.

The force at each contact is assumed to obey Coulomb’s friction law, which states that the contact force must lie within a cone during rolling contact and must lie on the boundary of the cone in the direction that dissipates the most energy during sliding. Since sliding is a function of body velocities, the following kinematic relationship will be needed:

$$\dot{q} = G(q)\nu \quad (3)$$

where  $G$  depends on the specific orientation parameterization used for three-dimensional systems and is the identity matrix for planar systems.

Equation (3) provides a connection between the distance functions and the matrix  $W_n$  as follows:  $W_n^T = \frac{\partial \psi_n}{\partial q} G$ . Note that one can define analogous (local) tangential displacement functions  $\psi_t$  and  $\psi_o$  with elements  $\psi_{it}$  and  $\psi_{io}$  for which the following hold:  $W_t^T = \frac{\partial \psi_t}{\partial q} G$  and  $W_o^T = \frac{\partial \psi_o}{\partial q} G$ .

Coulomb’s friction law requires that the contact force remain within a cone. When the contact is rolling, the contact force may lie anywhere inside the cone, but when the contact is sliding, the contact force must be one that maximizes energy dissipation. For  $\lambda_{in} \geq 0$ , let  $\mathcal{F}_i(\mu_i, \lambda_{in})$  denote the friction cone at contact  $i$ :

$$\mathcal{F}_i(\mu_i, \lambda_{in}) = \{(\lambda_{it}, \lambda_{io}) : \mu_i^2 \lambda_{in}^2 - \lambda_{it}^2 - \lambda_{io}^2 \geq 0\} \quad (4)$$

where  $\mu_i$  is the coefficient of friction acting at contact  $i$ .

Next, define orthogonal sliding velocity components  $v_{it}$  and  $v_{io}$ . The vectors of sliding velocities for all the contacts are:  $v_t = W_t^T \nu + \frac{\partial \psi_t}{\partial t}$  and  $v_o = W_o^T \nu + \frac{\partial \psi_o}{\partial t}$  with  $i^{\text{th}}$  elements  $v_{it} = W_{it}^T \nu + \frac{\partial \psi_{it}}{\partial t}$  and  $v_{io} = W_{io}^T \nu + \frac{\partial \psi_{io}}{\partial t}$ , respectively. Then Coulomb’s law at contact  $i$  may be written as follows:

$$(\lambda_{it}, \lambda_{io}) \in \arg \max_{(\lambda_{it}, \lambda_{io}) \in \mathcal{F}_i} (-\lambda_{it} v_{it} - \lambda_{io} v_{io}), \quad (5)$$

which has a useful equivalent formulation [17]:

$$0 = \mu_i \lambda_{in} (W_{it}^T \nu + \frac{\partial \psi_{it}}{\partial t}) + \lambda_{it} \sigma_i \quad (6)$$

$$0 = \mu_i \lambda_{io} (W_{io}^T \nu + \frac{\partial \psi_{io}}{\partial t}) + \lambda_{io} \sigma_i \quad (7)$$

$$0 \leq \sigma_i \perp \mu_i^2 \lambda_{in}^2 - \lambda_{it}^2 - \lambda_{io}^2 \geq 0 \quad (8)$$

where  $\sigma_i$  is a Lagrange multiplier arising from the conversion of the maximum dissipation condition from its ‘‘argmax’’ form into the inequality form given above. Note that at a solution of these conditions,  $\sigma_i = \sqrt{v_{it}^2 + v_{io}^2}$ , which is the magnitude of the slip rate at contact  $i$ .

Compactly, Coulomb’s law for all contacts is:

$$0 = (U \lambda_n) \circ (W_t^T \nu + \frac{\partial \psi_t}{\partial t}) + \lambda_t \circ \sigma \quad (9)$$

$$0 = (U \lambda_n) \circ (W_o^T \nu + \frac{\partial \psi_o}{\partial t}) + \lambda_o \circ \sigma \quad (10)$$

$$0 \leq \sigma \perp (U \lambda_n) \circ (U \lambda_n) - \lambda_t \circ \lambda_t - \lambda_o \circ \lambda_o \geq 0 \quad (11)$$

where  $U$  is the diagonal matrix with  $i^{\text{th}}$  diagonal element equal to  $\mu_i$  and  $\circ$  connotes the Hadamard product.

Some of the above equations are nonlinear in the unknowns (forces, configuration, and velocity), so their direct use in a time-stepping scheme would require the solution of (mixed) nonlinear complementarity problems (NCPs). In order to obtain a scheme based on (mixed) LCPs, a piecewise linear approximation of the quadratic friction cone with nonnegative force variables is needed (see figure 1). Let  $n_d$  friction force direction vectors  $d_j$  be chosen such that they positively span the space of possible friction forces, and let  $(\lambda_{if})_j$  be the friction force components in those directions. Also, let  $(\psi_{if}(q, t))_j$  be the corresponding (local) tangential displacement function. Then the equilibrium equation can be approximated as:

$$0 = W_n(q) \lambda_n + W_f(q) \lambda_f + f(q, t) \quad (12)$$

where  $\lambda_f \in \mathbb{R}^{n_c n_d}$  has  $n_c$  elements  $\lambda_{if} \in \mathbb{R}^{n_d}$  with elements  $(\lambda_{if})_j$ , the vector  $\psi_f \in \mathbb{R}^{n_c n_d}$  is defined analogously, and  $W_f^T = \frac{\partial \psi_f}{\partial q} G$ .

The approximate friction cone can be represented as:

$$\bar{\mathcal{F}}_i(\mu_i, \lambda_{in}) = \{ \lambda_{if} \mid \mu_i \lambda_{in} - e^T \lambda_{if} \geq 0, \lambda_{if} \geq 0 \} \quad (13)$$

where  $e \in \mathbb{R}^{n_d}$  is vector of ones. Let  $v_{if} = [(v_{if})_1 \dots (v_{if})_{n_d}]^T = \frac{\partial \psi_{if}}{\partial q} G \nu = W_{if}^T \nu$  be the vector of components of the sliding velocity at contact  $i$  in the friction directions. The approximate version of the dissipation condition becomes:

$$\lambda_{if} \in \arg \max_{\lambda_{if} \in \bar{\mathcal{F}}_i} (-\lambda_{if}^T W_{if}^T \nu). \quad (14)$$

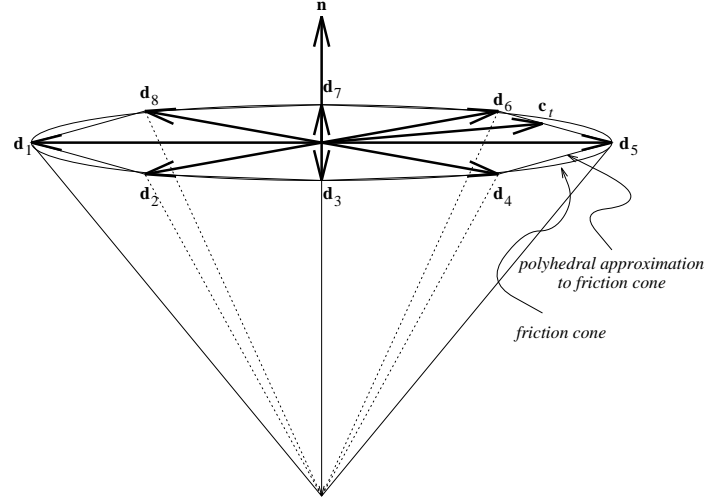


Figure 1: Friction cone approximated by an eight-sided pyramid defined by friction direction vectors  $d_j$ .

Reusing the slack variable  $\sigma_i$  (with slightly different meaning now), a useful equivalent LCP formulation of the maximum dissipation condition for the approximate friction cone is:

$$0 \leq \lambda_{if} \perp W_{if}^T \nu + e \sigma_i + \frac{\partial \psi_{if}}{\partial t} \geq 0 \quad (15)$$

$$0 \leq \sigma_i \perp \mu_i \lambda_{in} - e^T \lambda_{if} \geq 0, \quad (16)$$

where now  $\sigma_i$  approximates the sliding speed at contact  $i$ . Maximum dissipation for all contacts can be written compactly as:

$$0 \leq \lambda_f \perp W_f^T \nu + E \sigma + \frac{\partial \psi_f}{\partial t} \geq 0 \quad (17)$$

$$0 \leq \sigma \perp U \lambda_n - E^T \lambda_f \geq 0 \quad (18)$$

where  $E$  is the block diagonal matrix with  $i^{\text{th}}$  block on the main diagonal given by  $e$ .

To summarize, there are two models of interest which differ only in their descriptions of the friction cone.

**Model-IQC** (quadratic cones): equations (1-3,9-11).

**Model-ILC** (linear cones): equations (2,3,12,17,18).

### 3 Discrete-time models

A desirable outcome for any time-stepping scheme is that its solution at the end of each time step of the discrete-time model equals the (continuous) solution of the instantaneous-time model at the same time. Typically however, computational efficiency and/or convergence issues

force one to design a scheme that does not exactly meet this outcome. To prepare for the design of a time-stepper that solves a linear problem for each time step, the quadratic friction cone was approximated by a piecewise linear cone. In the following, two time-stepping schemes will be presented. The unknowns for both are the configuration vector, contact forces, and sliding speeds at the end of the time step.

Let  $t^\ell$  and denote the time at which one has a solution and let  $t^{\ell+1} = t^\ell + h$  denote the time at which one would like an estimate of the solution (the term  $h$  is called the step size). To eliminate  $\nu$ ,  $\dot{q}$  can be approximated using a backward Euler formula as follows:

$$\Delta q = q^{\ell+1} - q^\ell = G(q)v^{\ell+1}h \quad (19)$$

where  $q^\ell = q(t^\ell)$ . Note that since  $\Delta q$  is in the range of  $G$  (see equation (3)), the following useful identity holds:  $\Delta q = GG^T \Delta q$ .

### 3.1 A mildly nonlinear model: Model-DQC

After substituting equation (19) into **Model-IQC**, and replacing all occurrences of the variables  $(q, \lambda_n, \lambda_t, \lambda_o, \sigma)$  with their values at the end of the time step,  $(q^{\ell+1}, \lambda_n^{\ell+1}, \lambda_t^{\ell+1}, \lambda_o^{\ell+1}, \sigma^{\ell+1})$ , all model equations are nonlinear in the unknowns.

To remove some of the nonlinearities from the time-stepper, let  $W_n, W_t, W_o, G$ , and  $f$  be evaluated at  $q^\ell$ . In addition, let the distance function vector be approximated by the linear terms in its Taylor series expansion:

$$0 \leq \lambda_n^{\ell+1} \perp W_n^T G^T q^{\ell+1} + b_n \geq 0 \quad (20)$$

where  $b_n = \psi_n^\ell + \frac{\partial \psi_n^\ell}{\partial t} h - W_n^T G^T q^\ell$ . Now the only remaining nonlinearities are the quadratic terms in Coulomb's law. The result is a mildly nonlinear discrete-time model, **Model-DQC**. For each time step, the NCP composed of equations (1,20) and the following must be solved:

$$0 = (U\lambda_n) \circ (W_t^T G^T q + b_t) + \lambda_t \circ \sigma h \quad (21)$$

$$0 = (U\lambda_n) \circ (W_o^T G^T q + b_o) + \lambda_o \circ \sigma h \quad (22)$$

$$0 \leq \sigma \perp (U\lambda_n) \circ (U\lambda_n) - \lambda_t \circ \lambda_t - \lambda_o \circ \lambda_o \geq 0 \quad (23)$$

where the variables  $q, \lambda_n, \lambda_t, \lambda_o$ , and  $\sigma$  appearing in equations (21-23) are to be evaluated at time  $t^{\ell+1}$ ,  $b_t = \frac{\partial \psi_t^\ell}{\partial t} h - W_t^T G^T q^\ell$  and  $b_o = \frac{\partial \psi_o^\ell}{\partial t} h - W_o^T G^T q^\ell$ .

#### Summary of Model-DQC:

For each time step, solve mixed NCP of size  $n_q + 4n_c$  defined by equations (1,20-23).

### 3.2 A linear model: Model-DLC

The other discrete-time model of interest, **Model-DLC** can be derived from **Model-ILC** by the same procedure. The result is a mixed LCP defined as follows:

$$\begin{pmatrix} 0 \\ \rho_n^{\ell+1} \\ \rho_f^{\ell+1} \\ s^{\ell+1} \end{pmatrix} = B \begin{pmatrix} q^{\ell+1} \\ \lambda_n^{\ell+1} \\ \lambda_f^{\ell+1} \\ \sigma^{\ell+1} \end{pmatrix} + b \quad (24)$$

$$0 \leq \begin{pmatrix} \rho_n^{\ell+1} \\ \rho_f^{\ell+1} \\ s^{\ell+1} \end{pmatrix} \perp \begin{pmatrix} \lambda_n^{\ell+1} \\ \lambda_f^{\ell+1} \\ \sigma^{\ell+1} \end{pmatrix} \geq 0 \quad (25)$$

where

$$B = \begin{pmatrix} 0 & W_n & W_f & 0 \\ W_n^T G^T & 0 & 0 & 0 \\ W_f^T G^T & 0 & 0 & E \\ 0 & U & -E^T & 0 \end{pmatrix}, \quad b = \begin{pmatrix} f \\ b_n \\ b_f \\ 0 \end{pmatrix}, \quad (26)$$

$b_n$  is defined as above, and  $b_f = \frac{\partial \psi_f^\ell}{\partial t} h - W_f^T G^T q^\ell$ .

#### Summary of Model-DLC:

For each time step, solve mixed LCP of size  $n_q + (2+n_d)n_c$  defined by equations (24,25).

## 4 Uniqueness

The theorem presented here is the first known solution uniqueness result for general quasistatic multibody systems with dry friction. It applies only to the discrete-time models, **Model-DQC** and **Model-DLC**. Because of space limitations, the results are presented without proof, but these will be available in [4].

Before stating the result, the friction force components can be written as the following functions of the normal force component and the relative tangential displacement components  $\Delta_{it} = W_{it}^T G^T q^{\ell+1} + b_{it}$  and  $\Delta_{io} = W_{io}^T G^T q^{\ell+1} + b_{io}$ :

$$\begin{aligned} \lambda_{it} &= -\mu_i \lambda_{in} \frac{\Delta_{it}}{\sqrt{\Delta_{it}^2 + \Delta_{io}^2}} \\ \lambda_{io} &= -\mu_i \lambda_{in} \frac{\Delta_{io}}{\sqrt{\Delta_{it}^2 + \Delta_{io}^2}} \end{aligned} \quad (27)$$

where when  $\Delta_{it} = \Delta_{io} = 0$ , the fractions appearing in equation (27) are both equal to 0/0, and are taken to be a suitable pair of scalars  $(\alpha, \beta)$  such that  $\alpha^2 + \beta^2 \leq 1$ .

For given  $\{\mu_i \lambda_{in}\}_{i=1}^{n_c}$ , consider the following convex, nondifferentiable optimization problem in the variable  $q^{\ell+1}$ :

$$\begin{aligned} \min \quad & -f^T q^{\ell+1} + \sum_{i=1}^{n_c} \mu_i \lambda_{in} \sqrt{\Delta_{it}^2 + \Delta_{io}^2} \\ \text{s.t:} \quad & W_n^T G^T q^{\ell+1} + b_n \geq 0 \end{aligned} \quad (28)$$

where recall that  $\Delta_{it}$  and  $\Delta_{io}$  are functions of  $q^{\ell+1}$ . The physical interpretation of this problem is that the displacement of the system is one that avoids penetration while minimizing the work done against external and frictional forces. In other words, the system is “lazy” and so moves no more than it absolutely must.

The following result describes the precise connection between the above optimization problem (28) and the discrete-time model **Model-DQC**.

**Theorem 1** *If  $(q^{\ell+1}, \lambda_n, \lambda_t, \lambda_o)$  solves **Model-DQC** then  $q^{\ell+1}$  is a globally optimal solution to (28) corresponding to  $\lambda_n$ . Conversely, if  $q^{\ell+1}$  is a globally optimal solution to (28) for a given  $\lambda_n$  and if  $\lambda_n$  is equal to an optimal Karush-Kuhn-Tucker (KKT) multiplier of the constraint in (28), then defining  $(\lambda_t, \lambda_o)$  by (27), the tuple  $(q^{\ell+1}, \lambda_n, \lambda_t, \lambda_o)$  solves **Model-DQC**.*

A question relevant to the design of fixed-point time stepping schemes is whether or not the convex optimization problem (28) has a unique solution, for fixed  $\{\mu_i \lambda_{in}\}_{i=1}^{n_c}$ . Let  $(q^{\ell+1}, \lambda_n, \lambda_t, \lambda_o)$ , solve **Model-DQC**. Denote by  $dq$  a small change in  $q^{\ell+1}$ , and define the index sets:

$$\mathcal{I} \equiv \{i : \psi_{in} = 0 < \lambda_{in}\} \quad (29)$$

$$\mathcal{J} \equiv \{i : \psi_{in} = 0 = \lambda_{in}\}. \quad (30)$$

**Proposition 1** *Corresponding to the solution  $(q^{\ell+1}, \lambda_n, \lambda_t, \lambda_o)$  of **Model-DQC**,  $q^{\ell+1}$  is the unique solution of (28) if and only if the following implication holds:*

$$\left. \begin{aligned} W_{in}^T G^T dq &\geq 0, \quad i \in \mathcal{I} \cup \mathcal{J} \\ W_{it}^T G^T dq &= 0, \quad i \in \mathcal{I} \\ W_{io}^T G^T dq &= 0, \quad i \in \mathcal{I} \\ (f)^T dq &\geq 0 \end{aligned} \right\} \Rightarrow dq = 0. \quad (31)$$

Finally, consider an alternative model where the quadratic friction cone at each contact  $i$  is replaced by a four-sided linearized cone:

$$\{(\lambda_{it}, \lambda_{io}) : \max(|\lambda_{it}|, |\lambda_{io}|) \leq \mu_i \lambda_{in}\}. \quad (32)$$

In this case, instead of (27), we have

$$\begin{aligned} \lambda_{it} &= -\mu_i \lambda_{in} \frac{\Delta_{it}}{|\Delta_{it}|} \\ \lambda_{io} &= -\mu_i \lambda_{in} \frac{\Delta_{io}}{|\Delta_{io}|}. \end{aligned} \quad (33)$$

Moreover, a result similar to Theorem 1 holds with the optimization problem (28) replaced by the following linear program:

$$\begin{aligned} \min \quad & -f^T q^{\ell+1} + \sum_{i=1}^{n_c} \mu_i \lambda_{in} (|\Delta_{it}| + |\Delta_{io}|) \\ \text{s.t:} \quad & W_n^T G^T q^{\ell+1} + b_n \geq 0 \end{aligned} \quad (34)$$

where again recall that  $\Delta_{it}$  and  $\Delta_{io}$  are functions of  $q^{\ell+1}$ .

## 5 Example: fence-particle problem

Consider the problem of manipulating a particle (shown as a finite disc) of mass  $m$  initially at rest on a horizontal plane (the  $(x, y)$ -plane in Figure 2). The configuration of this system is  $q = [x_p \ y_p \ z_p]^T$ , where  $z_p$  is the height of the particle above the plane (of the page). The wall on the right is parallel to the  $(y, z)$ -plane (perpendicular to the plane of the page) and of infinite extent. The fence is parallel to the wall, of infinite extent, and can translate in the  $x$ - and  $y$ -directions, but cannot translate in the  $z$ -direction or rotate.<sup>1</sup> The vector of noncontact and noninertial forces  $f = [0 \ 0 \ -mg]^T$  is the gravitational force which acts in the negative  $z$ -direction.

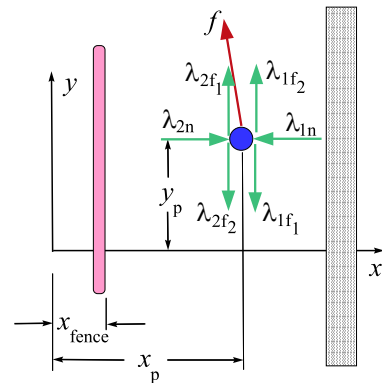


Figure 2: Schematic of fence-particle system.

<sup>1</sup>The latter constraint is to simplify the problem making the particle remain within the  $(x, y)$ -plane.

The three nonpenetration constraints,  $\psi_n(q, t) = [\psi_{1n}(q) \ \psi_{2n}(q, t) \ \psi_{3n}(q)]^T$  are written as:

$$\psi_{1n} = 1 - x_p \geq 0 \quad (35)$$

$$\psi_{2n} = x_p - x_{\text{fence}}(t) \geq 0 \quad (36)$$

$$\psi_{3n} = z_p \geq 0. \quad (37)$$

The corresponding lagrange multipliers are the normal components of the contact forces,  $\lambda_n(q, t) = [\lambda_{1n} \ \lambda_{2n} \ \lambda_{3n}]^T$ . Even though as shown, the particle is not in contact with the fence or wall on the right, the components of the corresponding contact forces are shown.<sup>2</sup> The possible contact force components between the particle and the plane are not shown.

In this example, solution uniqueness will be explored for two different friction laws for the contact between the particle and the  $(y, z)$ -plane: no friction and quadratic friction. An interesting point, is that for dynamic systems, the absence of friction guarantees solution existence and uniqueness of the predicted motion (not necessarily uniqueness of the contact forces) and the inclusion of friction leads to motion nonuniqueness. In the quasistatic system studied here, the reverse is true. For the case of linearized friction, the quadratic cone will be approximated by a four-sided friction pyramid (see Figure 3). The vari-

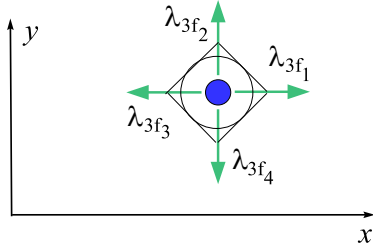


Figure 3: Friction direction vectors between the particle and the  $(x, y)$ -plane.

ous friction direction vectors at the three potential contacts imply the following definitions of the local tangential dis-

<sup>2</sup>Since translation in the  $z$ -direction is not possible in this problem, friction forces can act only in the plane of motion of the particle. This is why there are only two friction force directions for contacts 1 and 2.

placement functions:

$$(\psi_{1f})_1 = -y_p \quad (38)$$

$$(\psi_{1f})_2 = y_p \quad (39)$$

$$(\psi_{2f})_1 = y_p - y_{\text{fence}}(t) \quad (40)$$

$$(\psi_{2f})_2 = -y_p + y_{\text{fence}}(t) \quad (41)$$

$$\psi_{3t} = (\psi_{3f})_1 = x_p \quad (42)$$

$$\psi_{3o} = (\psi_{3f})_2 = y_p \quad (43)$$

$$(\psi_{3f})_3 = -x_p \quad (44)$$

$$(\psi_{3f})_4 = -y_p. \quad (45)$$

where  $y_{\text{fence}}(t)$  is the vertical position of the fence.

The various submatrices appearing in the matrix  $B$  are:

$$W_n = \begin{pmatrix} -1 & 1 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 1 \end{pmatrix} \quad U = \begin{pmatrix} \mu_1 & 0 & 0 \\ 0 & \mu_2 & 0 \\ 0 & 0 & \mu_3 \end{pmatrix} \quad (46)$$

$$W_f = \begin{pmatrix} 0 & 0 & 0 & 0 & 1 & 0 & -1 & 0 \\ -1 & 1 & 1 & -1 & 0 & 1 & 0 & -1 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \end{pmatrix} \quad (47)$$

$$E = \begin{pmatrix} 1 & 1 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 1 & 1 & 1 \end{pmatrix}. \quad (48)$$

Also, since the particle is a point mass, the matrix  $G$  is simply the identity matrix of size 3.

Other matrices for the nonlinear problem are

$$W_t = \begin{pmatrix} 0 & 0 & 1 \\ -1 & 1 & 0 \\ 0 & 0 & 0 \end{pmatrix} \quad W_o = \begin{pmatrix} 0 & 0 & 0 \\ 0 & 0 & 1 \\ 1 & 1 & 0 \end{pmatrix} \quad (49)$$

The time-dependent functions needed to define the vectors  $b_n, b_t, b_o, b_f$  were chosen as:

$$x_{\text{fence}}(t) = 0.5 + 0.4\sin(t) \quad (50)$$

$$y_{\text{fence}}(t) = t \quad (51)$$

With these choices, the fence translates in the  $y$  direction while oscillating in the  $x$ -direction without ever hitting the wall.

## 5.1 Results

Various values of the problem data were chosen to illustrate the theorems given in section 4. One common aspect of these problems is that the only forces that can act in the  $z$ -direction are the gravitational force and the normal component of the contact force between the particle and the  $(x, y)$ -plane. This implies that  $\lambda_{3n} = mg > 0$  and  $\psi_{3n} = 0$ .

### 5.1.1 Results: Model-DLC, no friction

The frictionless example of **Model-DLC** emphasizes why friction may be needed for solution uniqueness in quasistatic systems. Looking back at Proposition 1, the 2<sup>nd</sup> and 3<sup>rd</sup> rows of implication (31) are vacuous in the absence of friction. The removal of these equalities from the implication is what allows the construction of  $dq \neq 0$  still satisfying the two remaining inequalities of the implication. The Proposition requires us to look at a solution and then consider variations,  $dq$  from  $q^{\ell+1}$ . Assume a solution of the LCP with contact between the particle and the  $(x, y)$ -plane, but not with the wall or fence. In this case, we have  $W_{in}^T = [0 \ 0 \ 1]$  and  $f^T = [0 \ 0 \ -mg]$ . Let  $dq = [dx \ dy \ dz]^T$ . Then the inequalities of the implication yield  $dz = 0$ , but  $dx$  and  $dy$  are unconstrained. Therefore, there exist  $dq \neq 0$  satisfying the inequalities, and the implication fails. Thus the solution of  $q^{\ell+1}$  **Model-DLC** is not unique. In this particular case, the possible  $q^{\ell+1}$  solving **Model-DLC** are all those for which the particle remains in contact with the  $(x, y)$ -plane, and between the wall and fence. This conclusion was observed in practice. Specifically, the solution obtained was dependent on the initial guess.

### 5.1.2 Results: Model-DQC

Consider a solution for the system when the particle is not touching the fence or wall and the quadratic friction law is in effect at the contact with the  $(x, y)$ -plane. In this case, the matrices  $W_t$  and  $W_o$  are given as follows:

$$W_t = \begin{pmatrix} 1 \\ 0 \\ 0 \end{pmatrix}, \quad W_o = \begin{pmatrix} 0 \\ 1 \\ 0 \end{pmatrix}, \quad (52)$$

and  $W_n$  and  $f$  are as in the frictionless case.

From the frictionless case, it is known that the first and last rows of the implication constrain the  $z$ -component of  $dq$  to 0. Now, with friction present, the equations in the implication require the  $x$ - and  $y$ -components of  $dq$  to be zero also. Since the implication holds, by theorem 1 all  $q^{\ell+1}$  are unique, and in this case, if over the course of a time step, the fence will not reach the particle, then the particle will not move.

Consider a solution in which the particle is in contact with the fence. In this case, the matrices  $W_t$  and  $W_o$  gain rows, but do not change the conclusion - the motion of the particle is unique.

## 6 Summary

Two instantaneous-time models of three-dimensional quasistatic multibody systems with Coulomb friction have

been presented along with two corresponding discrete-time models. The discrete-time models take the form of complementarity problems for which led to the first known uniqueness results for such systems. A simple example was used to highlight a somewhat unexpected finding. In particular, dynamic multibody systems have unique accelerations when the friction coefficients are small enough. Whereas, for some quasistatic systems, the absence of friction can lead to nonunique system motions.

## Acknowledgements

The authors would like to thank Guanfang Liu and Jinglai Shen for insightful discussions on the ideas in this paper.

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